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## DESIGN OF AUTOMATIC TYRE INFLATION SYSTEM

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### Abstract

*Driven by studies that show that a drop in tire pressure by just a few PSI can result in the reduction of gas mileage, tire life, safety, and vehicle performance, we have developed an automatic, self-inflating tire system that ensures that tires are properly inflated at all times. Our design proposes and successfully implements the use of a centralized compressor that will supply air to all four tires via hoses and a rotary joint fixed between the wheel spindle and wheel hub at each wheel. The rotary joints effectively allow air to be channeled to the tires without the tangling of hoses. With the recent oil price hikes and growing concern of environmental issues, this system addresses a potential improvement in gas mileage; tire wear reduction; and an increase in handling and tire performance in diverse conditions. In this paper we have taken into consideration design aspects of the ATIS.*

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### KEY WORDS:

Automatic , self-inflating , human life , potential improvement .

### INTRODUCTION

In ancient time, after the discovery of wheel by man, it has been used extensively for various purposes and it is vital part of human life for ages. These wheels runs human life faster and faster with new technology and one such technology is on board air inflation system used in automobiles. Tyre is the second-highest cost for the automobile industry. This leads us to the condition that we should have regular attention towards the tyre of the vehicle.

This system(ATIS) helps to keep free from regular attention towards tyres and hence it reduces time of the person. Luxury cars and military vehicles are already using this system but the car of a common man don't have this system. Due to its low cost of production, a normal person can afford to have it installed in his car.

### CONCEPT

Automatic Tyre Inflation System simply works on the same principle as onboard or stationary tyre inflationsyaytem works. This system uses compressor to get the air from atmosphere, compress it and deliver it to the tyre for inflation. It has given the name automatic because it automatically checks the tyre pressure using the pressure gauge fitted there and if tyre is underinflated then the compressor starts to

deliver air to inflate the tyre. The switching of the circuit will take place using electronic circuits.

## DESIGN OBJECTIVES

### 1.Ability to Provide Proper Tire Pressure

The ideal functional objective of our design is its capability to adjust the pressures in all four tires of a passenger vehicle to obtain the proper pressure for varying road/driving conditions.

### 2. Ability to Provide Automatic System

A third objective is to provide all of the said benefits to the user through an automatic system, thus minimizing user intervention. Specifically, it is desired that the system automatically increase the tire pressures for the given road conditions. However, since this objective is closely linked with the ideal objectives in maintaining the proper tire pressure, and thus unattainable due to time constraints, this objective will not be pursued.

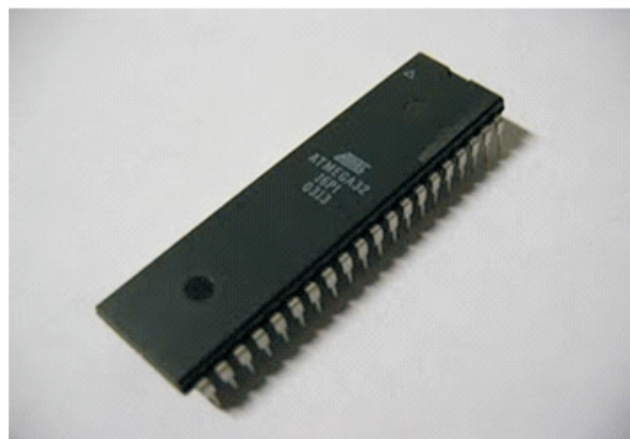
### 3 Low Cost Device

For both the customer and end user (vehicle owner), it is imperative to keep the price of the device as low as possible.. Considering the potential benefits and cost savings that this design has to offer and the prices of optional equipment for passenger vehicles with similar complexity, the target price range for this device has been identified as Rs 3000 - 5000. This is the price for both the OEM and vehicle owner, assuming that the OEM does not mark up the price. In addition, this price range should be able to support the costs of components of the system, manufacturing, and any necessary installation.

## ELECTRONIC REQUIREMENTS

### 1. Microcontroller ATMEGA 16:

ATmega16 is an 8-bit high performance microcontroller of Atmel's Mega AVR family with low power consumption. Atmega16 is based on enhanced RISC (Reduced Instruction Set Computing) architecture with 131 powerful instructions. Most of the instructions execute in one machine cycle. Atmega16 can work on a maximum frequency of 16MHz.



### ATMEGA 16 devices are available in 40-pin

It is 8-bit Microcontroller  
System is RISC Architecture  
It has Small set of Instruction set  
It has 131 powerful Instructions

Compatibility avail 28/40 Pin Ics  
 Operating Speed Max 16 MHz, Voltage 2-5.5 v  
 Memory: Flash Program-16KB, RAM-1 KB, EEPROM Data Mem- 512 Bytes  
 Low power, High speed Flash/EEPROM Technology  
 It has on chip Timers. 2 Timers are avail  
 It has in built Analog to Digital Converter, USART, Analog Comparator, SPI JTAG e  
 In built Multiplexer availability for signal Selection  
 It has serial as well as Parallel Communication facilities  
 In built Capture, Compare and Pulse width modulation  
 It has four 8 bit Ports designated as PORT A, PORT B, PORT C, PORT D for Internal and External usage.

### 2. L293D Motor Driver IC:-

L293D IC is basically used for driving the inductive loads like DC motors, stepper motors, and relays. It is a 16 pin DIP IC .It will simply amplify the logical input combinations from the microcontroller IC to drive the inductive loads like DC motor in our case.

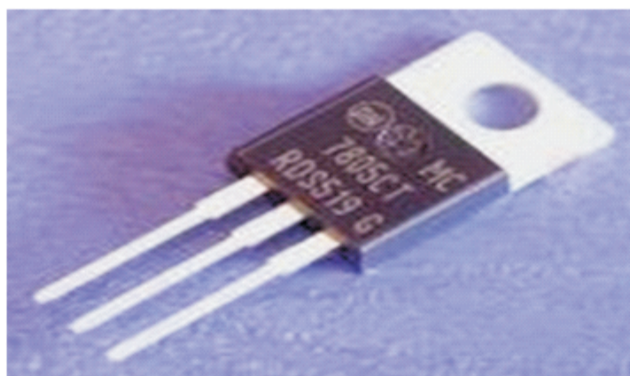
With the help of L293D IC we can drive two motors simultaneously at a time. It has four I/P pins and four O/P pins for controlling the devices by using microcontroller. For a single motor the combinations of two I/P are used for taking the I/P from the microcontroller and after amplification the corresponding two O/P combinations are connected with motor.



### 3. 7805 Voltage Regulator IC:-

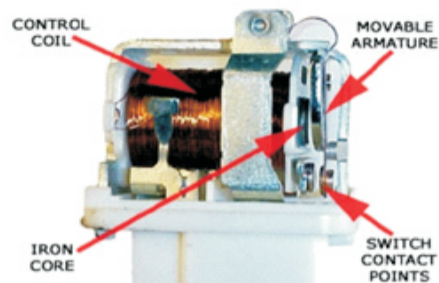
It is a three pin IC used as a voltage regulator. It converts unregulated DC current into regulated DC current.

It has given name as 7805, in which last two digits 05 certainly shows that it converts the given voltage into 05V.



### 4. RELAY SIGNAL CONTROLLER:-

Relays are simple switches which are operated both electrically and mechanically. Relays consist of an electromagnet and also a set of contacts. The switching mechanism is carried out with the help of the electromagnet. The main operation of a relay comes in places where only a low-power signal can be used to control a circuit. It is also used in places where only one signal can be used to control a lot of circuits.



### Engineering Analysis:-

#### 1. Complete V-Belt Design :-

Using Design Data Book By B. D. Shiwalkar

Given: Rated Torque,  $T_R = 40 \text{ kg-cm}$

$$T_R = 4 \text{ N-m}$$

Speed Of Motor,  $N_1 = 1000 \text{ rpm}$

Speed Of Wheel,  $N_2 = 333.33 \text{ rpm}$

#### For Rated Power ( $P_R$ ):

$$\begin{aligned} P_R &= 2\pi N_1 * T_R / 60 \\ &= 2\pi * 1000 * 4 / 60 \\ P_R &= 418.879 \text{ W} = 0.419 \text{ KW} \end{aligned}$$

#### For Design Power ( $P_d$ ):

$$P_d = P_R * K_1$$

Where  $K_1$  = Load Factor

For Electric Motor to Line Shaft

$$K_1 = 1.10$$

$$P_d = 0.419 * 1.10$$

$$P_d = 0.461 \text{ KW}$$

Now for selection of belt designation.

As  $P_d$  Lies in Range i.e. 0.35-3.5 KW

We Select Belt Designation As 'A'.

For 'A' Type Belt (Table no.)

Nominal Width,  $W = 13 \text{ mm}$  or  $\frac{1}{2} \text{ in}$

Nominal Thickness,  $t = 8 \text{ mm}$  or  $\frac{5}{16} \text{ in}$

Recommended min. Pulley dia.,  $D_1 = 75 \text{ mm}$  or  $3 \text{ in}$

Max. No. Of strands = 6

Bending Stress Factor,  $K_b = 17.6 * 10^3$

Centrifugal Tension Factor,  $K_c = 2.52$

#### Now For Pitch Line Velocity, ( $V_p$ )

$$\begin{aligned} V_p &= \pi D_1 * N_1 / 60 \\ &= \pi * 75 * 1000 / 60 * 1000 \\ V_p &= 3.927 \text{ m/sec} = 235.61 \text{ m/min} \end{aligned}$$

**Now For Dia. Of Driven Pulley,  $D_2$** 

In 'V' Belt Slip is Negligible Because of Wedging Action

So,

$$\pi D_2 * N_2 = \pi D_1 * N_1$$

$$D_2 = 75 * 1000 / 333.33$$

$$D_2 = 225 \text{ mm} = 9 \text{ in}$$

**Now For Belt Tensions,  $F_1$  &  $F_2$** 

$$F_1 - F_2 = P_d / V_p = 0.461 * 103 / 3.927$$

$$F_1 - F_2 = 117.392 \text{ N} \quad \dots (1)$$

Again We Have,

$$F_1 / F_2 = e^{\mu \theta \operatorname{cosec}(\alpha/2)}$$

Where,  $\mu$  = Coefficient of Friction = 0.3 usually

$$\theta = \text{Angle of Lap on Smaller Pulley} = \pi - (D_1 - D_2) / C$$

$C$  = Centre Distance Between Pulleys

$$= D_1 + D_2 = 300 \text{ mm}$$

$$\theta = \pi - (225 - 75) / 300$$

$$\theta = 2.642 \text{ rad}$$

$$\alpha = \text{Groove Angle} = 38^\circ$$

$$F_1 / F_2 = e^{0.3 * 2.642 * \operatorname{cosec}(38/2)} = 11.41$$

$$F_1 = 11.41 * F_2$$

Therefore Putting This Value in eq. (1) We Get,

$$F_1 = 128.671 \text{ N}$$

$$F_2 = 11.277 \text{ N}$$

**Now For Power Rating Per Belt,**

$$W = (F_w - F_c) * (e^{\mu \theta \operatorname{cosec}(\alpha/2)} - 1 / e^{\mu \theta \operatorname{cosec}(\alpha/2)}) * V_p$$

Where,

$$F_w = \text{Working Load} = (\text{Nominal Width})^2 = 13^2$$

$$= 169 \text{ N}$$

$$F_c = \text{Centrifugal Tension} = K_c * (V_p / 5)^2$$

$$= 2.52 * (3.927 / 5)^2$$

$$= 1.554 \text{ N}$$

$$W = (169 - 1.554) * (e^{0.3 * 2.642 * \operatorname{cosec}(38/2)} - 1 / e^{0.3 * 2.642 * \operatorname{cosec}(38/2)}) * 3.927$$

$$W = 412.976 \text{ N} = 0.413 \text{ KW}$$

Therefore,

$$\text{No. Of Strands} = P_d / \text{Power Rating}$$

$$= 1.116 = 1$$

But Max. No. Of Strands = 6

Therefore Design Is Safe

Now for Length of Belt (L),

$$L = \pi/2 * (D_1 + D_2) + 2C + (D_1 - D_2)^2 / 4C$$

$$= \pi/2 * (225 + 75) + 2 * 300 + (75 - 225)^2 / 4 * 300$$

$$L = 1089.989 \text{ mm} = 1.09 \text{ m}$$

Now For Larger Pulley Design,  
 For 'A' Designation Belt,  
 Length Of Pitch,  $l_p=11$  mm  
 Pitch Width,  $b=3.3$  mm  
 Pitch Height,  $h=8.7$  mm  
 Pitch,  $e=15$  mm  
 Half Pitch,  $f=9$  to  $12$  mm

Min. Pitch Dia.,  $D_p=125$  mm For  $\alpha=38$

Width of Pulley Rim,  $b_p$   
 $b_p=(\text{No. of Strands}-1)*\text{Pitch}+2*\text{Half Pitch}$   
 $= (2-1)*15+2*11$   
 $b_p=22$  mm  
 For Other Pulley Details,

#### For Diameter $D_2=225$

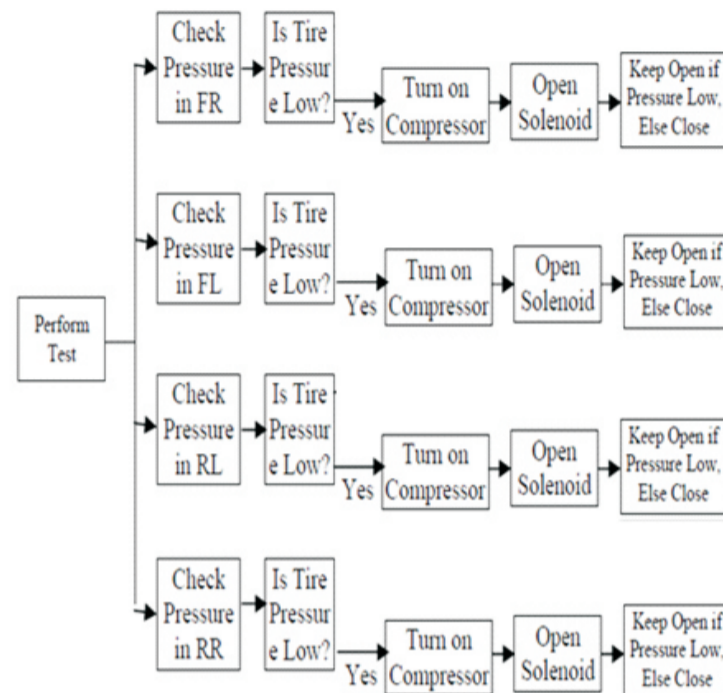
As  $D_2 > 150$  mm  
 So Arm Type Construction is Required  
 For  $D_2=225$  mm & Width= 22 mm  
 No. of Arms=4  
 No. of sets=1  
 Now For Design of Smaller Pulley,  
 Designation Parameter Are Same  
 For Dia.  $D_1=75$  mm  
 We Select Web Construction

#### 2. Speed Of Tyre:-

From Design Data Book By. Dr. B. D. Shiwalkar

Given, For V-Belt Drive  
 RPM of motor ( $N_1$ ) = 1000 rpm  
 Diameter of pulley on motor shaft ( $D_1$ ) = 3 inch  
 Diameter of pulley on wheel axle ( $D_2$ ) = 9 inch  
 Radius of tyre ( $R$ ) = 0.3 m.  
 From formula,  
 $N_1 D_1 = N_2 D_2$   
 $1000 * 3 = N_2 * 9$   
 $N_2 = 333.33$  rpm.  
 $V = (R * 2\pi N) / 60$   
 $= (0.3 * 2\pi * 333.33) / 60$   
 $= 10.472$  m/s.  
 Hence, Speed of tire = 37.7 km/hr.

#### Working Flow Diagram:-



### CONCLUSION:-

The dynamically-self-inflating tire system would be capable of succeeding as a new product in the automotive supplier industry. It specifically addresses the needs of the consumers by maintaining appropriate tire pressure conditions for:

- Reduced tire wear
- Increased fuel economy
- Increased overall vehicle safety

Because such a product does not currently exist for the majority of passenger vehicles, the market conditions would be favorable for the introduction of a self-inflating tire system. Through extensive engineering analysis, it has also been determined that the self-inflating tire system would actually function as desired. In particular, the product would be capable of:

- Providing sufficient airflow to the tire with minimal leakage
- Withstanding the static and dynamic loading exerted on the rotary joints.

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